

## Problem Set 8, PS8 due Monday June 7

**PS8-1** 4-61 Air at 300 kPa, and 200 C enters a nozzle at 30 m/sec and leaves at 100 Kpa and 180 m/sec. The nozzle has an inlet area of  $80 \text{ cm}^2$ . Determine a) the mass flow rate, b) the exit temperature of the air and, c) the exit area of the nozzle.

**PS8-2** 4-83 Steam at 10 MPa, 500 C enters an adiabatic turbine at 3 kg/sec and leaves the turbine at 20 kPa. The turbine produces 2 MW. Neglecting kinetic energy changes determine the temperature of the steam leaving the turbine

**PS8-3** 4-86 R-134a at a flow rate of 1.2 kg/sec is compressed in an adiabatic compressor from a saturated vapor at -20 C to .7 MPa and 70 C. Determine a) the power input to the compressor and, b) the volume flow rate at the compressor inlet

## 4-61

$$v_1 = \frac{RT}{p} = \frac{.287 \times (273.15 + 200)}{300 \text{ kPa}} = .4526 \text{ m}^3/\text{kg}$$

$$\text{a) } m = \frac{AV}{v} = \frac{80}{10^4} \times 30 \text{ m/sec} \times \frac{1}{.4526} = .5303 \text{ kg/sec}$$

b) open system

$$Q = 0, W = 0$$

$$h + \frac{V^2}{2} = \text{constant}$$

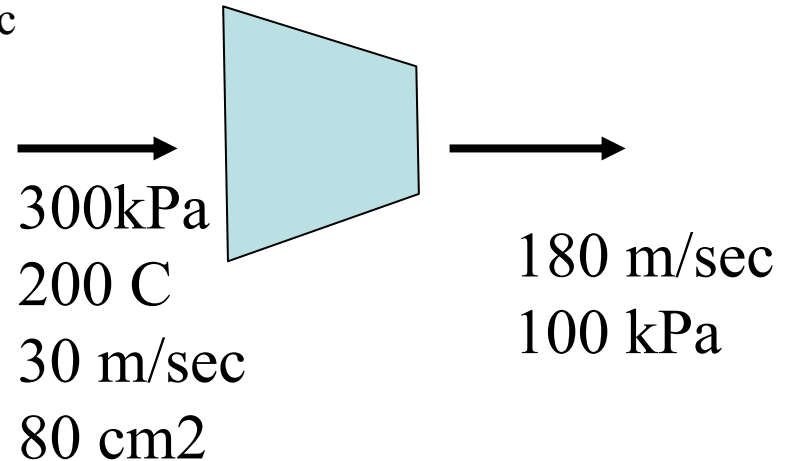
$$h_1 + \frac{V_1^2}{2} = h + \frac{V_2^2}{2}$$

$$c_p(T_1 - T_2) = \frac{V_2^2}{2} - \frac{V_1^2}{2}$$

$$(T_1 - T_2) = \frac{1}{c_p} \left( \frac{V_2^2}{2} - \frac{V_1^2}{2} \right)$$

$$(T_1 - T_2) = \frac{.5}{1.02} (180^2 - 30^2)$$

$$T_2 = 184.5^\circ\text{C}$$



$$v_2 = \frac{RT_2}{p_2} = \frac{.287 \times (273.15 + 184.5)}{100 \text{ kPa}}$$

$$v_2 = 1.313 \text{ m}^3/\text{kg}$$

$$m = \frac{A_2 V_2}{v_2}$$

$$A_2 = \frac{m \times v_2}{V_2} = \frac{1.313 \times .5303 \text{ kg/sec}}{180 \text{ m/sec}} = .00387 \text{ m}^2$$

4-83

$$h @ 10 \text{ MPa}, 500^\circ \text{C} = 3737.7 \text{ kJ/kg}$$

open system

region in space

$$W = m\Delta h$$

$$W = 2000 \text{ kJ/sec} = 2 \text{ kg/sec}(3737.7 - h_2)$$

$$h_2 = 3737.7 - \frac{2000}{2} = 2737.7 \text{ kJ/kg}$$

@  $h = 2707$ ,  $p = 2 \text{ kPa}$ ,  $T = 110.8^\circ \text{C}$  by interpolation

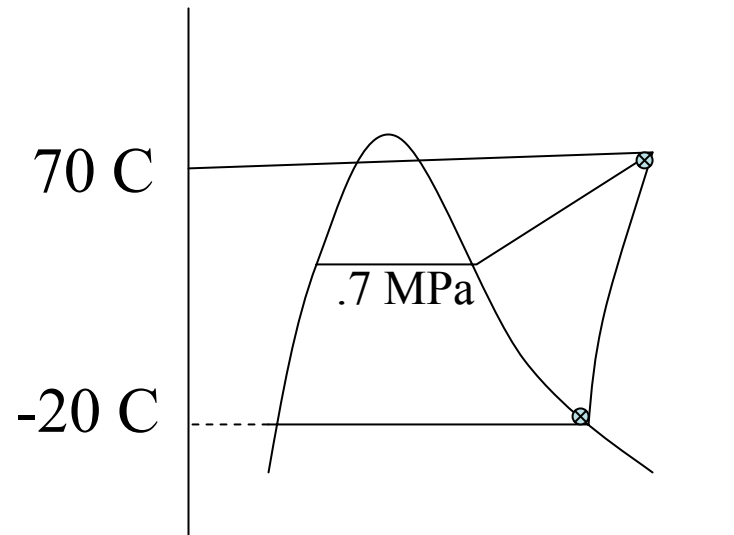
	.01kPa.	.02kPa	.05kPa
100	2687.5	<b>2686.5</b>	2682.5
<b>110.6</b>		2707	
200	2879.5	<b>2879.2</b>	2877.7

$$h @ .01 \text{ kPa}, 100^\circ \text{C} = 2687.5 - \frac{.02 - .01}{.05 - .01} \times (2687.5 - 2682.5) = \mathbf{2686.5}$$

$$h @ .05 \text{ kPa}, 200^\circ \text{C} = 2879.5 - \frac{.02 - .01}{.05 - .01} \times (2879.5 - 2877.7) = \mathbf{2879.2}$$

$$T @ .02 \text{ kPa}, h = 2707 = 100 + \left( \frac{2707 - 2686.5}{2879.2 - 2686.5} \right) \times 100 = 101.6^\circ \text{F}$$

4-86



@  $-20^{\circ}\text{C}$ ,  $v_g = .146 \text{ m}^3/\text{kg}$ ,  $h_g = 235.3 \text{ kJ/kg}$

@  $70^{\circ}\text{C}$ ,  $.7 \text{ MPa}$ ,  $h_g = 307. \text{ kJ/kg}$

open system

$$W = m\Delta h = 1.3 = 2\text{kg} (307. \text{ kJ/kg} - 235.3 \text{ kJ/kg})$$

$$W = 86.04 \text{ kJ/sec}$$

$$V = m v_1 = 1.2 \text{ kg} \times .146 \text{ m}^3/\text{kg} = .176 \text{ m}^3/\text{sec}$$