

FORCES

SURFACE FORCES: Forces in the fluid which act on a solid or imaginary surface. These forces are specified in terms of an intensity or stress(force per unit area).

Local: Pressure

Shear (friction) stress

Total or Resultant: Lift, Drag, etc.

BODY FORCES: Forces that act on the mass of the fluid. We will only deal with gravity. Body forces are given in terms of force per unit volume.

γ = Specific Weight = Weight/Volume
= lbs/cubic feet or newtons/cubic m

Force = γ x volume

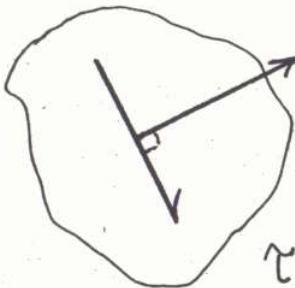
ρ = Density = mass/volume
= slugs/cubic ft or kgs/cubic m

$\gamma = \rho g$

STRESS

STRESS: Force per unit area of surface – N/m^2 .

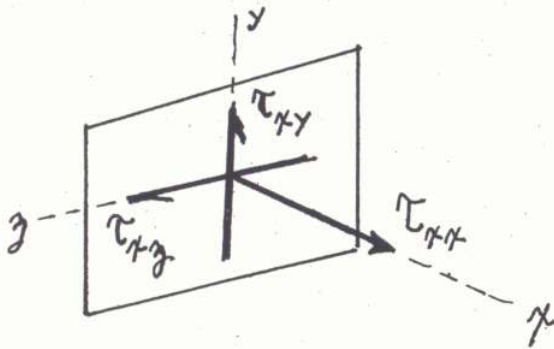
Can be normal and tangential components.



The diagram shows an irregularly shaped surface element. A line segment is drawn from a point on the surface, perpendicular to the surface, representing the normal direction. A right-angle symbol is shown at the point of contact. An arrow points along this normal line, labeled τ_m .

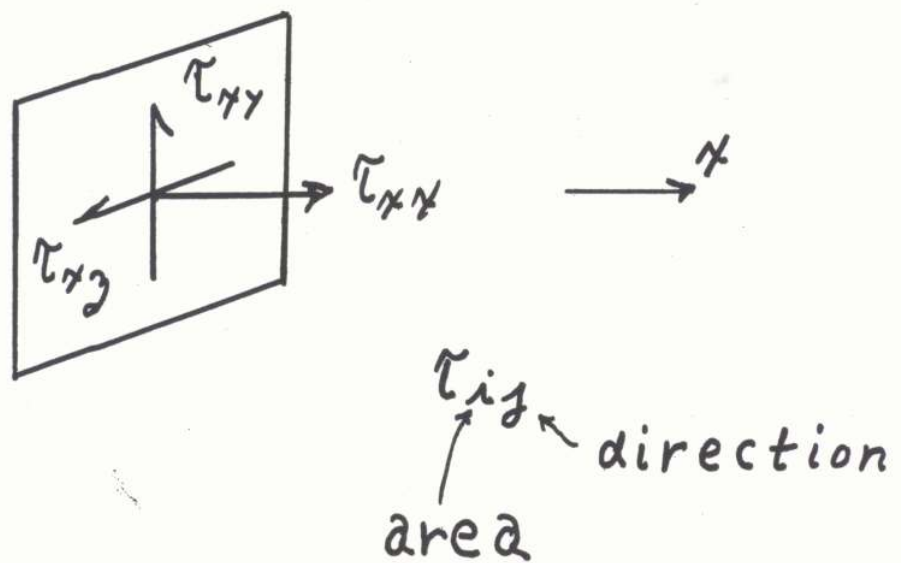
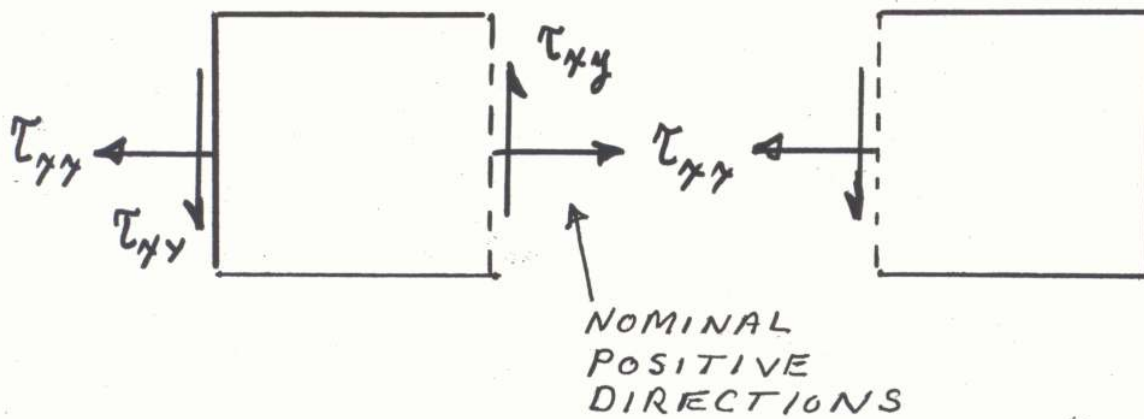
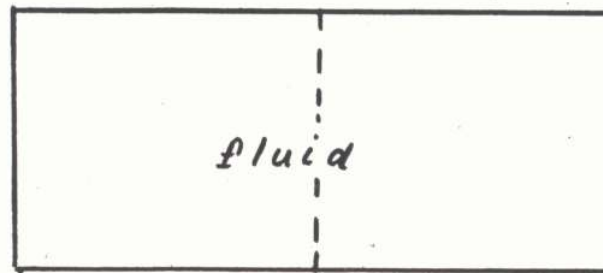
$$\tau_m = \lim_{\Delta A \rightarrow 0} \frac{\Delta F_m}{\Delta A} = \text{Normal}$$
$$\tau_s = \lim_{\Delta A \rightarrow 0} \frac{\Delta F_s}{\Delta A} = \text{Shear}$$

The normal component is unique. The tangent component requires an angle or can be given as two components.



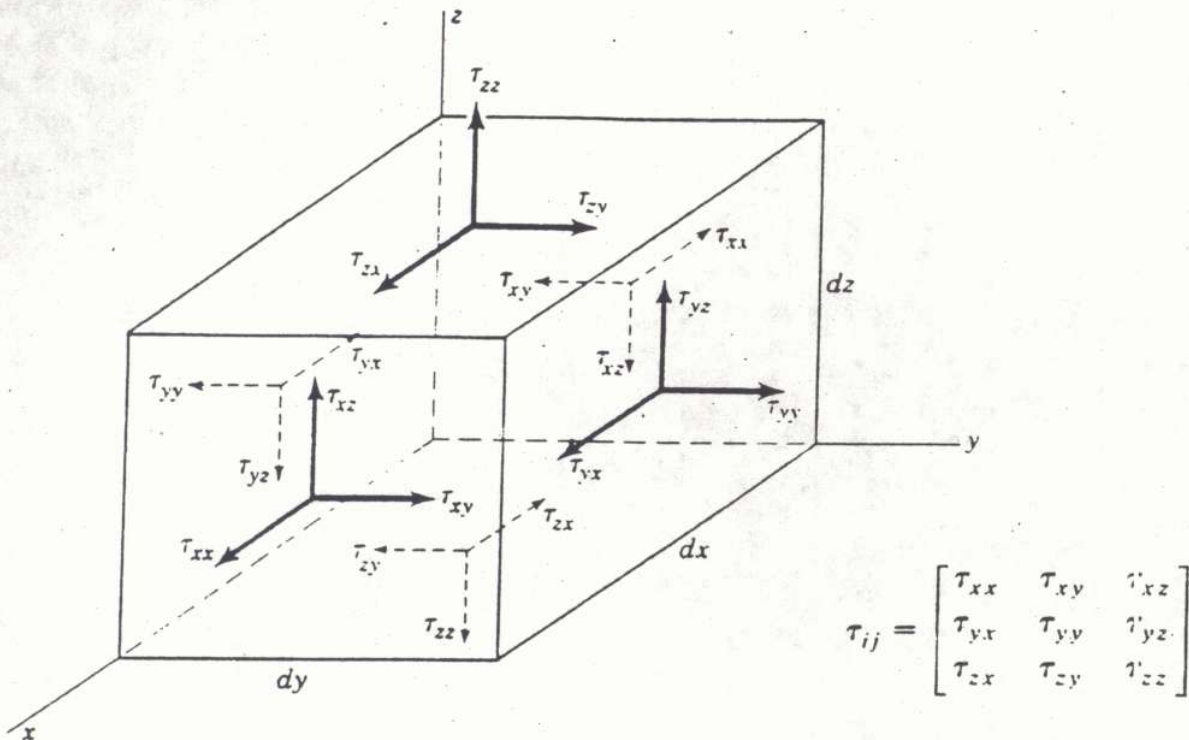
FOR A GIVEN ORIENTATION
THERE ARE THREE
INDEPENDENT COMPONENTS
OF STRESS.

STRESS NOTATION



STRESS TENSOR

STATE OF STRESS: In general requires three components (normal and two shears) on each of three mutually perpendicular surfaces for a total of nine components.



Stress is a Second Order Tensor, has nine components.

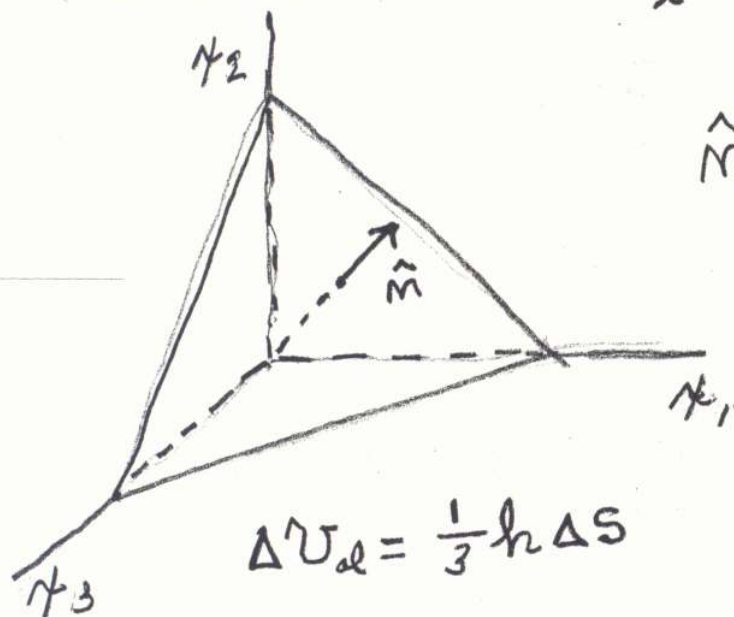
Vector: First Order Tensor, has three components.

Velocity: V_x, V_y, V_z

Scalar: Zero Order Tensor, has magnitude only.

Stress Vector

$$\vec{T}_i^{(m)}$$

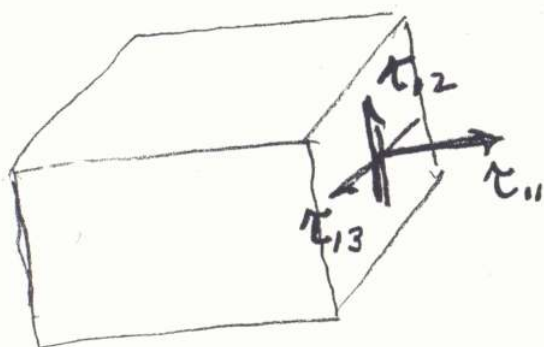


$$\hat{m} = m_1 \hat{e}_1 + m_2 \hat{e}_2 + m_3 \hat{e}_3$$

$$\Delta \vec{S} = \hat{m} \Delta S$$

$$\Delta S_1 = \Delta \vec{S} \cdot \hat{e}_1 = m_1 \Delta S$$

$$\Delta S_i = m_i \Delta S$$



$$\vec{T}^{(1)} = \sigma_{11} \hat{e}_1 + \sigma_{12} \hat{e}_2 + \sigma_{13} \hat{e}_3$$

$$\vec{T}^{(2)} =$$

$$\vec{T}^{(3)} =$$

$$\sum \vec{F} = \frac{D \vec{V}}{Dt} dm = \frac{D \vec{V}}{Dt} dm = e \vec{a} \Delta V$$

$$\vec{T}^{(m)} \Delta S - \vec{T}^{(1)} \Delta S_1 - \vec{T}^{(2)} \Delta S_2 - \vec{T}^{(3)} \Delta S_3 + e \Delta V \vec{g} = e \vec{a} \Delta V$$

$$\vec{T}^{(M)} = \vec{T}^{(1)} m_1 + \vec{T}^{(2)} m_2 + \vec{T}^{(3)} m_3 + \frac{1}{3} e h \vec{g}$$

$$= \frac{1}{3} e h \vec{a}$$

$$h \rightarrow 0$$

$$\vec{T}^{(M)} = \vec{T}^{(1)} m_1 + \vec{T}^{(2)} m_2 + \vec{T}^{(3)} m_3$$

$$= \vec{T}^{(k)} m_k$$

$$T_1^{(M)} = \sigma_{11} m_1 + \sigma_{21} m_2 + \sigma_{31} m_3$$

$$T_2^{(M)} = \sigma_{12} m_1 + \sigma_{22} m_2 + \sigma_{32} m_3$$

$$T_3^{(M)} = \sigma_{13} m_1 + \sigma_{23} m_2 + \sigma_{33} m_3$$

$$T_i^{(M)} = \sigma_{ji} m_j$$

Subject: Physics - The last word

The small book is a guide for teachers and students

For more information, please contact the publisher

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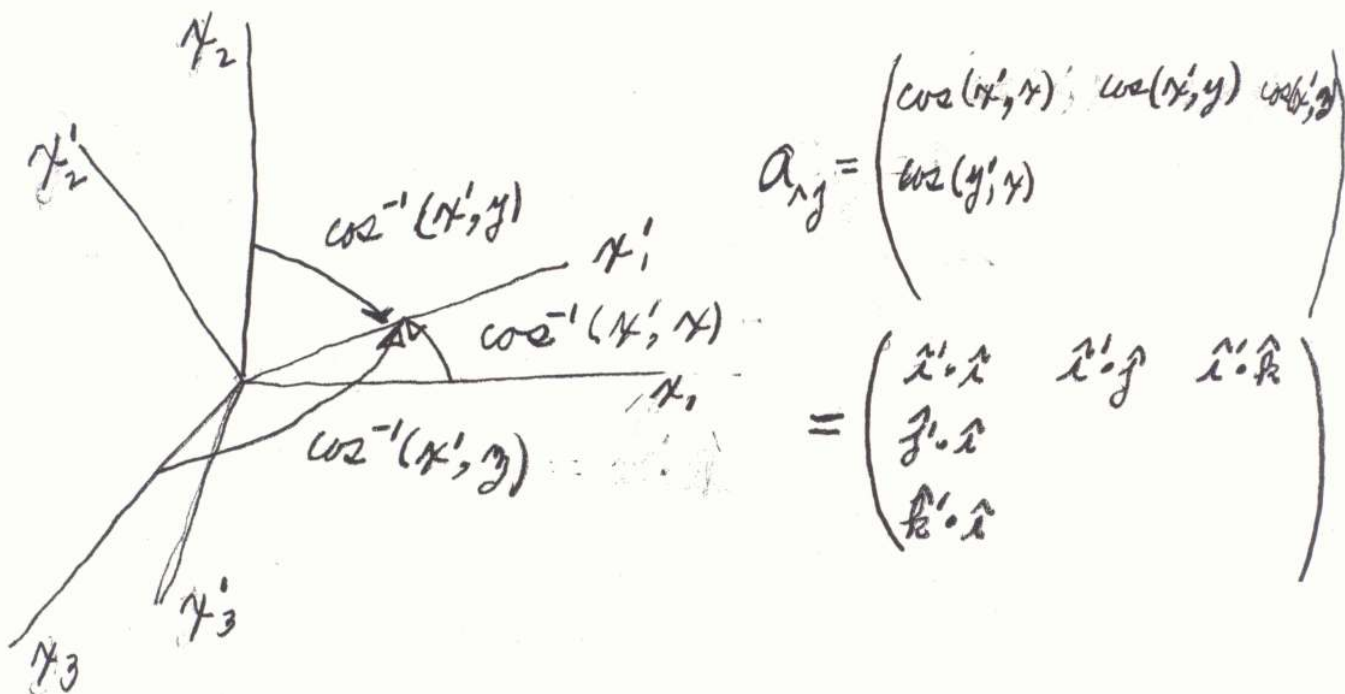
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Subject: Physics - The last word

Teachers and students

Rotation of Coordinate axes



$$x'_1 = x_1 a_{11} + x_2 a_{12} + x_3 a_{13}$$

$$x'_2 =$$

$$x'_i = a_{ij} x_j$$

$$T_{ij}^{(m)} = m_j \sigma_{ji} \quad T_{ij}^{(x'_i)} = a_{ij} \sigma_{ji}$$

$$T_{ij}^{(x'_i)} \cdot \hat{x}'_1 = \tau_{x'_1 x'_1} = a_{11} a_{1j} \sigma_{ji} = \tau'_{11}$$

$$T_{ij}^{(x'_i)} \cdot \hat{x}'_2 = \tau_{x'_1 x'_2} = a_{12} a_{1j} \sigma_{ji} = \tau'_{12}$$

$$T_{ij}^{(x'_i)} \cdot \hat{x}'_3 = \tau_{x'_1 x'_3} = a_{13} a_{1j} \sigma_{ji} = \tau'_{13}$$

$$\tau'_{kl} = a_{kj} a_{il} \sigma_{ji}$$

Obeys Transformation Rule - $\therefore$ stress is a tensor

Stress Tensor

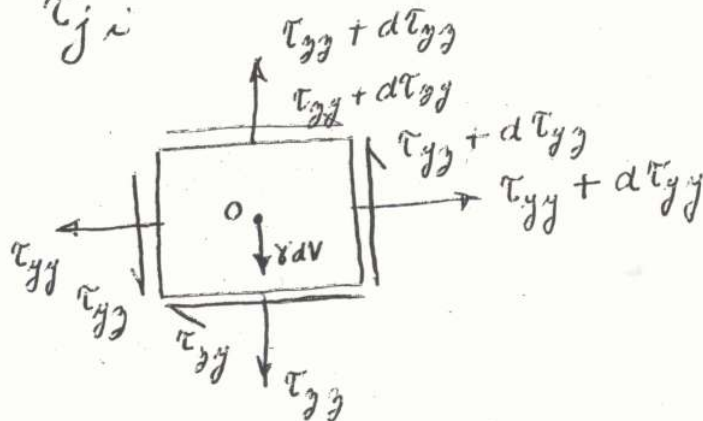
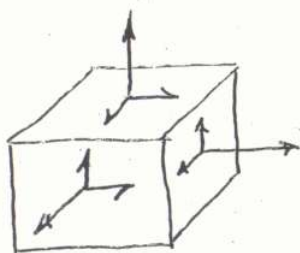
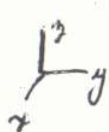
$$\tau_{ij} \Rightarrow \begin{pmatrix} \tau_{xx} & \tau_{xy} & \tau_{xz} \\ \tau_{yx} & \tau_{yy} & \tau_{yz} \\ \tau_{zx} & \tau_{zy} & \tau_{zz} \end{pmatrix}$$

These stresses do not represent the average stresses on the representative block faces, but the local stresses at a point.

It takes 9 quantities to specify the state of stress at a point, i.e., the three components on each of three mutually perpendicular planes.

⇒ The stress tensor is symmetric

$$\tau_{ij} = \tau_{ji}$$



$$\begin{aligned} \Sigma M_o &= \cancel{\frac{dx}{2} \tau_{yy} dx dy} + \frac{dx}{2} (\tau_{yy} + d\tau_{yy}) dx dy \\ &\quad - \cancel{\frac{dx}{2} \tau_{yy} dx dy} - \frac{dx}{2} (\tau_{yy} + d\tau_{yy}) dx dy \\ &= \frac{1}{12} e (dy^2 + dz^2) dx dy dz \alpha \end{aligned}$$

$$\Sigma M_o \sim r^2 dm \alpha$$

$$\tau_{yz} + \frac{1}{2} d\tau_{yz} - \tau_{zy} - \frac{1}{2} d\tau_{zy} = \frac{1}{12} e (dy^2 + dz^2) \alpha$$

$$\therefore \tau_{yz} = \tau_{zy}$$

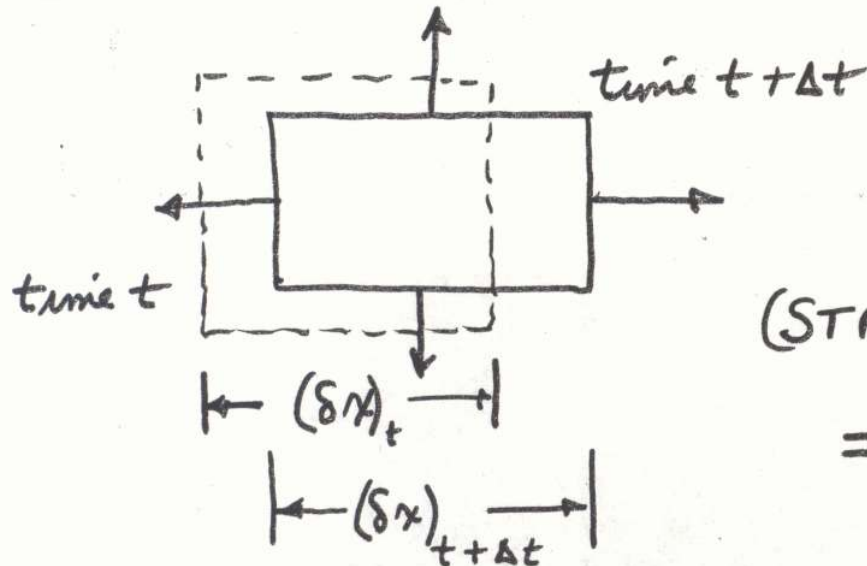
$$\text{Also } \tau_{xy} = \tau_{yx}$$

$$\tau_{xz} = \tau_{zx}$$

$$\tau_{ij} = \tau_{ji}$$

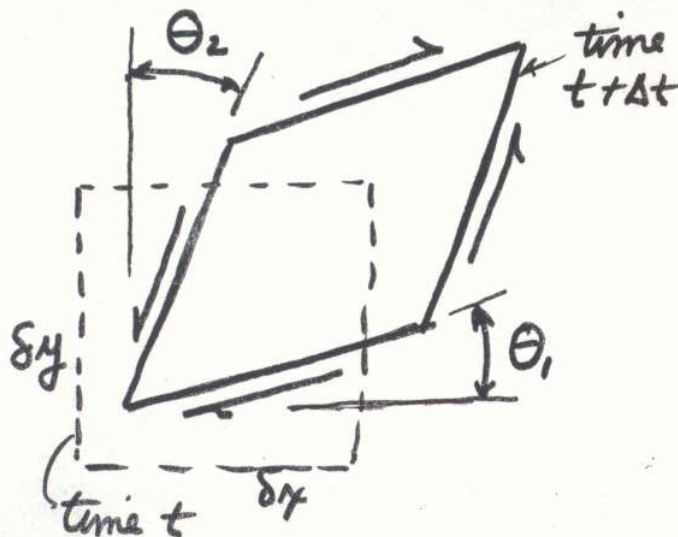
STRESS + STRAIN

NORMAL STRESS $\rightarrow$ ELONGATION



$$\begin{aligned} (\text{STRAIN})_x &= \epsilon_{xx} \\ &= \frac{(\delta x)_{t+\Delta t} - (\delta x)_t}{(\delta x)_t} \end{aligned}$$

SHEAR STRESS $\rightarrow$ CHANGE IN ANGLE

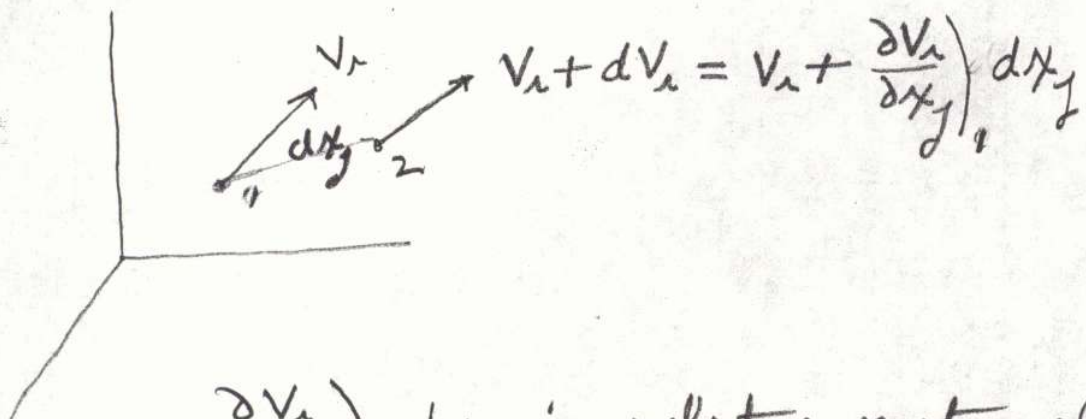


$$\begin{aligned} (\text{STRAIN})_{xy} &= \epsilon_{xy} \\ &= \Theta_1 + \Theta_2 \end{aligned}$$

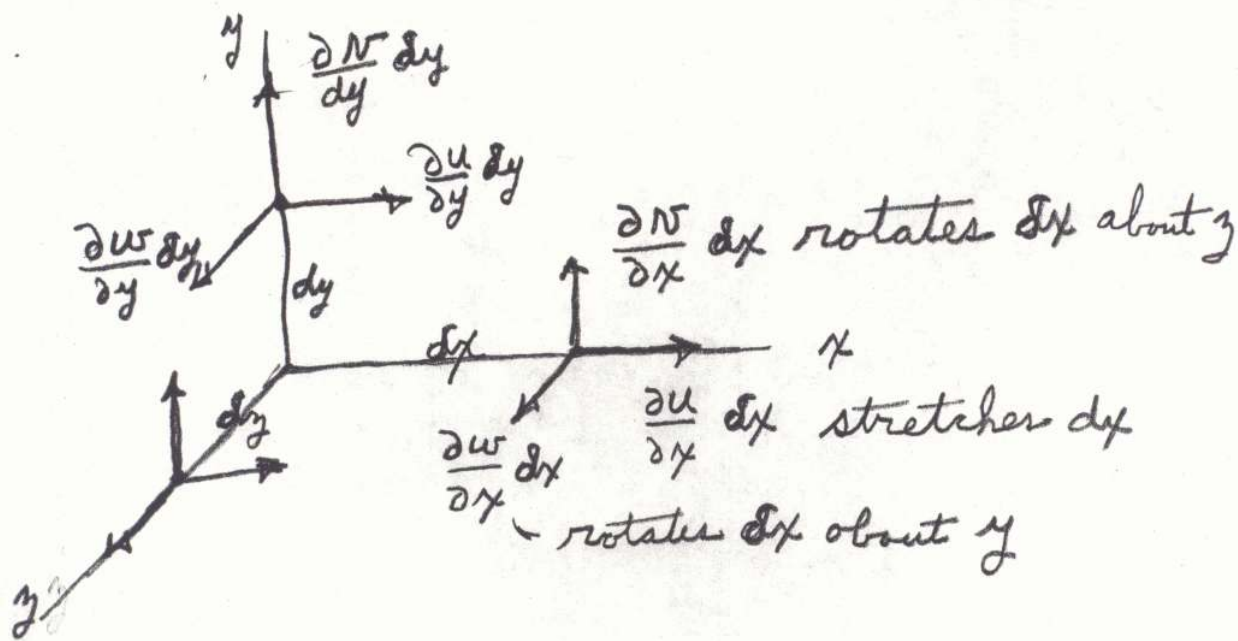
$$\tau_{xy} \propto \epsilon_{xy}$$

$$(\text{Rotation})_z = \frac{1}{2}(\Theta_1 - \Theta_2)$$

Strain



$\left(\frac{dV_1}{dx_j}\right) dx_j$ is relative motion of point 2 with respect to 1.



Normal Strain

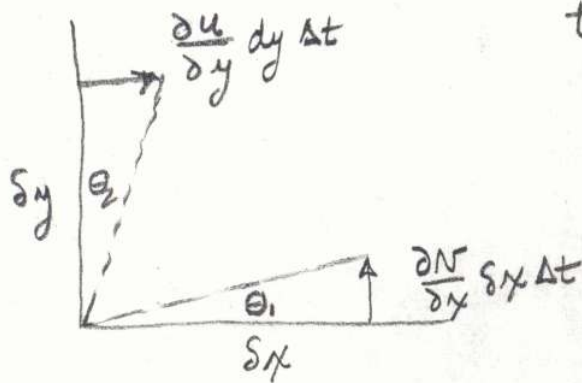
$$\epsilon_{xx} = \frac{(\delta x)_{t+\Delta t} - (\delta x)_t}{(\delta x)_t} = \frac{\frac{\partial u}{\partial x} \delta x \Delta t}{\delta x}$$

$$\dot{\epsilon}_{xx} = \frac{\partial u}{\partial x} \quad \text{strain rate}$$

$$\dot{\epsilon}_{yy} = \frac{\partial v}{\partial y} \quad \dot{\epsilon}_{zz} = \frac{\partial w}{\partial z}$$

$$\text{Vol strain rate} = \frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z} = \frac{\partial v}{\partial x} + \frac{\partial w}{\partial y}$$

Shear Strain



$$\tan \theta_1 \cong \theta_1 = \frac{\frac{\partial v}{\partial x} \delta x \Delta t}{\delta x}$$

$$\dot{\theta}_1 = \frac{\partial v}{\partial x}$$

$$\tan \theta_2 \cong \theta_2 = \frac{\frac{\partial u}{\partial y} \delta y \Delta t}{\delta y}$$

$$\dot{\theta}_2 = \frac{\partial u}{\partial y}$$

$$\text{Shear strain} = \theta_1 + \theta_2 = \frac{\partial v}{\partial x} + \frac{\partial u}{\partial y}$$

$$\text{strain}_{xy} = \frac{1}{2} \left(\frac{\partial v}{\partial x} + \frac{\partial u}{\partial y} \right) = \dot{\epsilon}_{xy}$$

$$\text{rot}_{xy} = \frac{1}{2} \left(\frac{\partial v}{\partial x} - \frac{\partial u}{\partial y} \right)$$

$$\dot{\epsilon}_{ij} = \frac{1}{2} \left(\frac{\partial N_i}{\partial x_j} + \frac{\partial N_j}{\partial x_i} \right) = \dot{\epsilon}_{ji}$$

$$\dot{\omega}_{ij} = \frac{1}{2} \left(\frac{\partial N_i}{\partial x_j} - \frac{\partial N_j}{\partial x_i} \right) = -\dot{\omega}_{ji}$$

$$\dot{\omega}_{11} = \dot{\omega}_{22} = \dot{\omega}_{33} = 0$$

$$\frac{\partial N_i}{\partial x_j} = \dot{\epsilon}_{ij} + \dot{\omega}_{ij}$$

Stress-Strain

Newtonian fluid $\Rightarrow$ stress linearly related to strain

$$\tau_{ij} = C_{ijkl} \epsilon_{kl} \quad 81 \text{ coefficients}$$

$$\epsilon_{kl} \text{ symmetric} \quad C_{ijkl} = C_{jilk}$$

$$\tau_{ij} \text{ symmetric} \quad C_{ijkl} = C_{jikl}$$

Fluid is isotropic = properties are independent of direction

$$C_{ijkl} = \text{isotropic tensor}$$

$$C_{ijkl} = \lambda \delta_{ij} \delta_{kl} + \mu (\delta_{ik} \delta_{jl} + \delta_{il} \delta_{jk}) \\ + \gamma (\delta_{ik} \delta_{jl} - \delta_{il} \delta_{jk})$$

$\gamma = 0$ to satisfy symmetry

$$\tau_{ij} = [\lambda \delta_{ij} \delta_{kl} + \mu (\delta_{ik} \delta_{jl} + \delta_{il} \delta_{jk})] \epsilon_{kl} \\ = \lambda \epsilon_{kk} \delta_{ij} + \mu (\epsilon_{ij} + \epsilon_{ji}) \\ = \lambda \frac{\partial u_k}{\partial x_k} \delta_{ij} + \mu \left(\frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i} \right)$$

$$\sigma_{ij} = -p \delta_{ij} + \tau_{ij} \\ \tau_{jj} = 0$$

$$\tau_{jj} = 3\lambda \frac{\partial u_k}{\partial x_k} + 2\mu \frac{\partial u_j}{\partial x_j} = 0 \Rightarrow \lambda = -\frac{2}{3}\mu$$

$$\therefore \tau_{ij} = \mu \left(\frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i} \right) - \frac{2}{3}\mu \frac{\partial u_k}{\partial x_k} \delta_{ij}$$

Momentum - Finite Volume

$$\Sigma \vec{F} = \frac{D}{Dt} \int \vec{v} dm$$

$$\frac{D}{Dt} \int \alpha dV = \frac{\partial}{\partial t} \int \alpha dV + \int \alpha \vec{v} \cdot \hat{n} ds$$

$$\alpha = e \vec{v}$$

$$\Sigma \vec{F} = \frac{\partial}{\partial t} \int e \vec{v} dV + \int \vec{v} e \vec{v} \cdot \hat{n} ds$$



$$\vec{T}^{(n)} \Delta s$$

$$\Sigma \vec{F} = \int \vec{T}^{(n)} ds$$

$$\int \vec{f} dV + \int \vec{f} dV$$

gravity

$$\vec{f} = e \vec{g}$$

Momentum - Differentiation

$$\sum \vec{F} = \frac{D}{Dt} \int \vec{v} dm$$

$$\begin{aligned} \frac{D}{Dt} \int \alpha dV &= \int \frac{\partial \alpha}{\partial t} dV + \int \alpha \vec{v} \cdot \hat{n} dS \\ &= \int \frac{\partial \alpha}{\partial t} dV + \int \nabla \cdot (\alpha \vec{v}) dV \end{aligned}$$

$$\alpha = \rho \vec{v}$$

$$\begin{aligned} \sum F_x &= \int T_x^{(n)} dS + \int f_x dV \\ &= \int \frac{\partial \sigma_{jx}}{\partial x_j} dV + \int f_x dV \end{aligned}$$

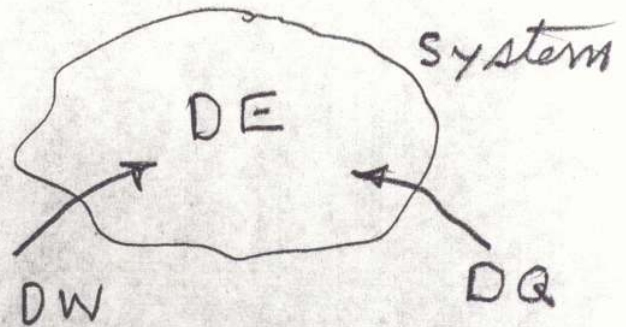
$$\int_V \left[\frac{\partial \rho v_x}{\partial t} + \frac{\partial \rho v_j v_x}{\partial x_j} - \frac{\partial \sigma_{jx}}{\partial x_j} - f_x \right] dV = 0$$

Since the volume is arbitrary, the integrand = 0

$$\begin{aligned} \frac{\partial \rho v_x}{\partial t} + \frac{\partial \rho v_j v_x}{\partial x_j} &= \rho \frac{\partial v_x}{\partial t} + \rho v_j \frac{\partial v_x}{\partial x_j} = \rho \frac{Dv_x}{Dt} \\ &= \frac{\partial \sigma_{jx}}{\partial x_j} + f_x \end{aligned}$$

Energy Equation

$$\frac{DE}{Dt} = \frac{DW}{Dt} + \frac{DQ}{Dt}$$



E = point function

W, Q = path functions

$$E = \int \left(e + \frac{v^2}{2} \right) \rho dV$$

$\nwarrow$ kinetic
 $\nearrow$ internal

$$DW = \int f_n V_n dt dV + \int T_n^{(m)} V_n dt dS$$

$$\frac{DW}{Dt} = \int f_n V_n dV + \int \sigma_{jn} m_j V_n dS$$

$$= \int f_n V_n dV + \int \frac{\partial \sigma_{jn} V_n}{\partial x_j} dV$$

$$\frac{DQ}{Dt} = - \int g_j m_j dS = - \int \frac{\partial g_j}{\partial x_j} dV$$

$\nwarrow$ Outward normal

$$\epsilon = e \left(e + \frac{v^2}{2} \right)$$

$$\begin{aligned} \frac{D\epsilon}{Dt} &= \int \frac{\partial \epsilon}{\partial t} dV + \int \epsilon \vec{v} \cdot \hat{n} dS \\ &= \int \frac{\partial \epsilon}{\partial t} dV + \int \nabla \cdot (\epsilon \vec{v}) dV \end{aligned}$$

$$\int \left[\frac{\partial \epsilon}{\partial t} + \frac{\partial \epsilon v_j}{\partial x_j} - f_n v_n - \frac{\partial \tau_{jn} v_n}{\partial x_j} + \frac{\partial g_j}{\partial x_j} \right] dV = 0$$

$$\begin{aligned} &\frac{\partial e(e + \frac{v^2}{2})}{\partial t} + \frac{\partial e v_j (e + \frac{v^2}{2})}{\partial x_j} \\ &= \frac{\partial \tau_{jn} v_n}{\partial x_j} + f_n v_n - \frac{\partial g_j}{\partial x_j} \\ &= e \frac{\partial (e + \frac{v^2}{2})}{\partial t} + e v_j \frac{\partial (e + \frac{v^2}{2})}{\partial x_j} = e \frac{D(e + \frac{v^2}{2})}{Dt} \end{aligned}$$

Mechanical Energy Equation

$$e \frac{Dv_n}{Dt} = \frac{\partial \tau_{jn}}{\partial x_j} + f_n = - \frac{\partial p}{\partial x_n} + \frac{\partial \tau_{jn}}{\partial x_j} + f_n$$

$$e \frac{Dv^2/2}{Dt} = \frac{\partial v_n \tau_{jn}}{\partial x_j} + v_n f_n - \tau_{jn} \frac{\partial v_n}{\partial x_j}$$

$$e \frac{Dv^2/2}{Dt} = \frac{\partial v_n \tau_{jn}}{\partial x_j} + v_n f_n + p \frac{\partial v_j}{\partial x_j} - \underbrace{\tau_{jn} \frac{\partial v_n}{\partial x_j}}_{\text{Dissipation of mechanical energy}}$$

$$e \frac{D\epsilon}{Dt} = - \underbrace{\frac{\partial q_i}{\partial x_j}}_{\text{heat transfer}} - \underbrace{p \frac{\partial V_j}{\partial x_j}}_{\substack{p dV \\ \text{work}}} + \underbrace{\tau_{yx} \frac{\partial V_x}{\partial x_j}}_{\text{dissipation}}$$

$\tau_{yx} \frac{\partial V_x}{\partial x_j}$ Dissipation of mechanical energy by friction $\Rightarrow$ appears as source term in the thermal energy equation and can increase e or result in a heat transfer.

$$\begin{aligned} \tau_{yx} \frac{\partial V_x}{\partial x_j} &= \left[\mu \left(\frac{\partial V_x}{\partial x_j} + \frac{\partial V_j}{\partial x_x} \right) - \frac{2}{3} \mu \frac{\partial V_k}{\partial x_k} \delta_{xy} \right] \frac{\partial V_x}{\partial x_j} \\ &= \left[2\mu \epsilon_{xy} - \frac{2}{3} \mu \frac{\partial V_k}{\partial x_k} \delta_{xy} \right] (\epsilon_{xy} + \omega_{xy}) \\ &= 2\mu \epsilon_{xy} \epsilon_{xy} - \frac{2}{3} \mu \left(\frac{\partial V_k}{\partial x_k} \right)^2 > 0 \end{aligned}$$

NAVIER STOKES EQUATIONS

$$\sigma_{ij} = -p \delta_{ij} + \tau_{ij}$$

$\uparrow$ average normal stress = pressure $\uparrow$ friction

Newtonian Fluid

$$\tau_{ij} \propto \text{Strain}_{ij}$$

Stokes Viscosity Law

$$\tau_{ij} = \mu \left(\frac{\partial v_i}{\partial x_j} + \frac{\partial v_j}{\partial x_i} \right) + \lambda \frac{\partial v_k}{\partial x_k} \delta_{ij}$$

$$\lambda = -\frac{2}{3}\mu \text{ so that } \frac{1}{3}\sigma_{jj} = -p$$

$$\rho \frac{Dv_i}{Dt} = \rho \left(\frac{\partial v_i}{\partial t} + v_j \frac{\partial v_i}{\partial x_j} \right)$$

$$= -\frac{\partial p}{\partial x_i} + \frac{\partial}{\partial x_j} \left[\mu \left(\frac{\partial v_i}{\partial x_j} + \frac{\partial v_j}{\partial x_i} \right) - \frac{2}{3}\mu \frac{\partial v_k}{\partial x_k} \delta_{ij} \right]$$

$$\tau_{xx} = \mu \left(\frac{4}{3} \frac{\partial u}{\partial x} - \frac{2}{3} \frac{\partial v}{\partial y} - \frac{2}{3} \frac{\partial w}{\partial z} \right)$$

$$\tau_{yy} = \mu \left(-\frac{2}{3} \frac{\partial u}{\partial x} + \frac{4}{3} \frac{\partial v}{\partial y} - \frac{2}{3} \frac{\partial w}{\partial z} \right)$$

$$\tau_{zz} = \mu \left(-\frac{2}{3} \frac{\partial u}{\partial x} - \frac{2}{3} \frac{\partial v}{\partial y} + \frac{4}{3} \frac{\partial w}{\partial z} \right)$$

$$\tau_{xy} = \tau_{yx} = \mu \left(\frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} \right)$$

$$\tau_{xz} = \tau_{zx} = \mu \left(\frac{\partial u}{\partial z} + \frac{\partial w}{\partial x} \right)$$

$$\tau_{yz} = \tau_{zy} = \mu \left(\frac{\partial v}{\partial z} + \frac{\partial w}{\partial y} \right)$$

SUMMARY

$$\frac{\partial \rho}{\partial t} + \frac{\partial \rho v_j}{\partial x_j} = 0$$

$$\rho \left(\frac{\partial v_i}{\partial t} + v_j \frac{\partial v_i}{\partial x_j} \right) = - \frac{\partial P}{\partial x_i} + \frac{\partial}{\partial x_j} \left[\mu \left(\frac{\partial v_i}{\partial x_j} + \frac{\partial v_j}{\partial x_i} \right) - \frac{2}{3} \mu \frac{\partial v_k}{\partial x_k} \right]$$

$$\rho c_v \left(\frac{\partial T}{\partial t} + v_j \frac{\partial T}{\partial x_j} \right) = - \frac{\partial}{\partial x_j} \left(k \frac{\partial T}{\partial x_j} \right) - p \frac{\partial v_j}{\partial x_j}$$

$$+ \mu \left[\left(\frac{\partial v_i}{\partial x_j} + \frac{\partial v_j}{\partial x_i} \right) \frac{\partial v_i}{\partial x_j} - \frac{2}{3} \mu \left(\frac{\partial v_j}{\partial x_j} \right)^2 \right]$$

dissipation

$p = \rho R T$ perfect gas

6 Equations

ρ, v_i, P, T unknown

Also c_v, k, μ are functions of temperature

Component equations

$$\begin{aligned} \rho \left(\frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} + w \frac{\partial u}{\partial z} \right) = & - \frac{\partial p}{\partial x} \\ & + \frac{\partial}{\partial x} \left[\mu \left(\frac{4}{3} \frac{\partial u}{\partial x} - \frac{2}{3} \frac{\partial v}{\partial y} - \frac{2}{3} \frac{\partial w}{\partial z} \right) \right] \\ & + \frac{\partial}{\partial y} \left[\mu \left(\frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} \right) \right] \\ & + \frac{\partial}{\partial z} \left[\mu \left(\frac{\partial u}{\partial z} + \frac{\partial w}{\partial x} \right) \right] \end{aligned}$$

SIMPLIFYING SITUATIONS

STEADY FLOW $\partial/\partial t = 0$

$$\frac{\partial \rho v_j}{\partial x_j} = 0$$

$$\rho v_j \frac{\partial v_i}{\partial x_j} = - \frac{\partial p}{\partial x_i} + \frac{\partial}{\partial x_j} \left[\mu \left(\frac{\partial v_i}{\partial x_j} + \frac{\partial v_j}{\partial x_i} \right) - \frac{2}{3} \mu \frac{\partial v_k}{\partial x_k} \delta_{ij} \right]$$

$$\rho c_p v_j \frac{\partial T}{\partial x_j} = \frac{\partial}{\partial x_j} \left(k \frac{\partial T}{\partial x_j} \right) - \rho \frac{\partial v_j}{\partial x_j} + \Phi$$

Could integrate unsteady equations to steady state. This is often done, however, it is usually faster to iterate steady equations to convergence if only steady state is desired.

INCOMPRESSIBLE FLOW

$\rho = \text{constant}$ ($M \leq 0.3$ in gases)

$$\frac{\partial v_j}{\partial x_j} = 0$$

$$\rho \left(\frac{\partial v_i}{\partial t} + v_j \frac{\partial v_i}{\partial x_j} \right) = - \frac{\partial p}{\partial x_i} + \frac{\partial}{\partial x_j} \left[\mu \left(\frac{\partial v_i}{\partial x_j} + \frac{\partial v_j}{\partial x_i} \right) \right]$$

$$\rho c_p \left(\frac{\partial T}{\partial t} + v_j \frac{\partial T}{\partial x_j} \right) = \frac{\partial}{\partial x_j} \left(k \frac{\partial T}{\partial x_j} \right) + \Phi$$

$$\mu = \mu(T), \quad k = k(T)$$

INCOMPRESSIBLE, CONSTANT PROPERTIES

$$\frac{\partial V_j}{\partial x_j} = 0$$

$$\frac{\partial V_i}{\partial t} + V_j \frac{\partial V_i}{\partial x_j} = - \frac{1}{\rho} \frac{\partial p}{\partial x_i} + \nu \frac{\partial^2 V_i}{\partial x_j \partial x_j}$$

$$\frac{\partial T}{\partial t} + V_j \frac{\partial T}{\partial x_j} = \alpha \frac{\partial^2 T}{\partial x_j \partial x_j}$$

$\nu = \mu / \rho$ kinematic viscosity

$\alpha = k / (\rho c_p)$ thermal diffusivity

INVISCID, ADIABATIC $\Rightarrow$ EULER EQUATIONS

$$\frac{\partial \rho}{\partial t} + \frac{\partial \rho V_j}{\partial x_j} = 0$$

$$\rho \left(\frac{\partial V_i}{\partial t} + V_j \frac{\partial V_i}{\partial x_j} \right) = - \frac{\partial p}{\partial x_i}$$

$$\rho c_p \left(\frac{\partial T}{\partial t} + V_j \frac{\partial T}{\partial x_j} \right) = - p \frac{\partial V_j}{\partial x_j}$$

Jameson, A. "Computational Aerodynamics for Aircraft Design" Science: Vol 245, July 28, 1989 pp 361-371

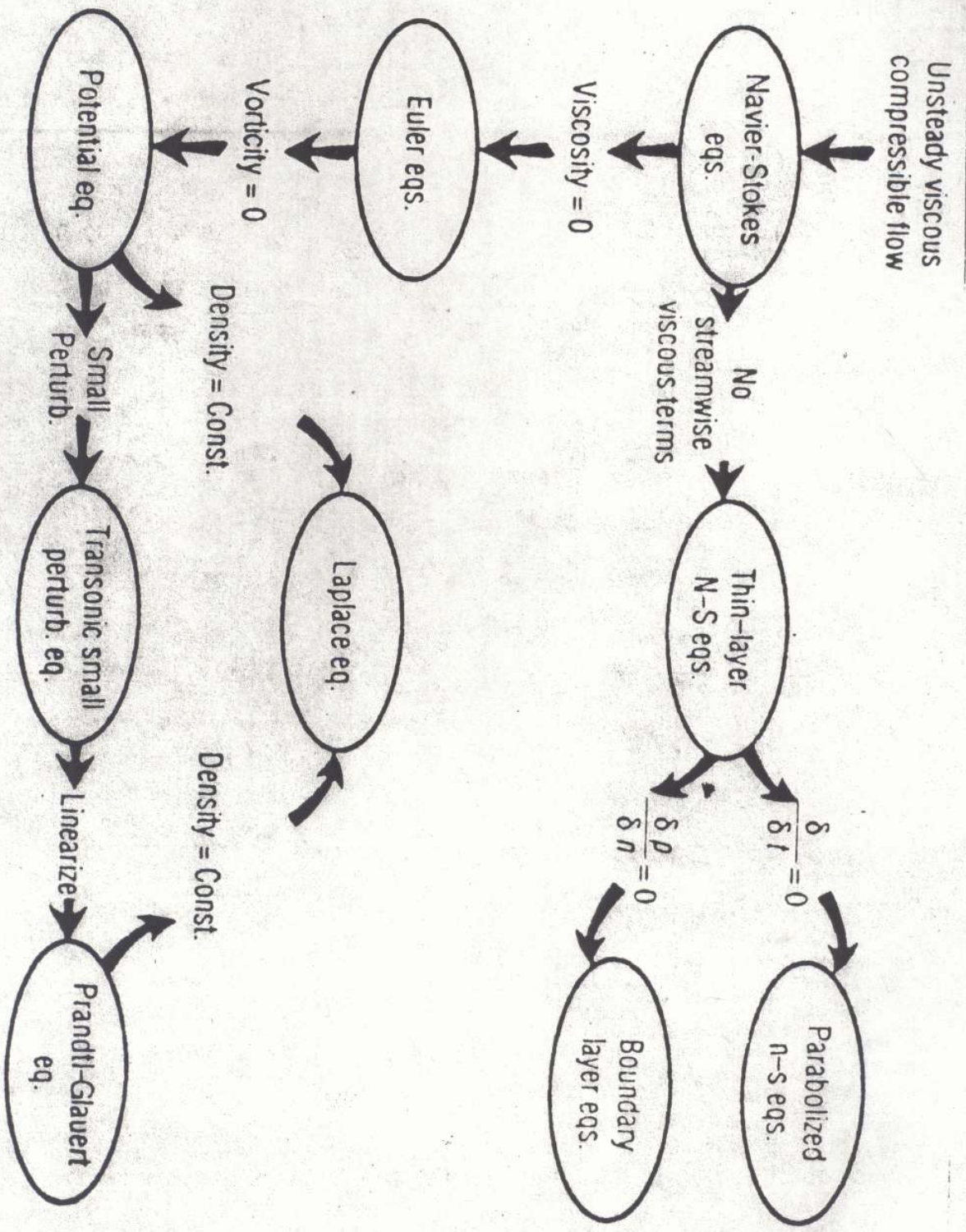


Fig. 2. Equations of fluid dynamics for mathematical models of varying complexity. (Supplied by Luis Miranda, Lockheed Corporation.)