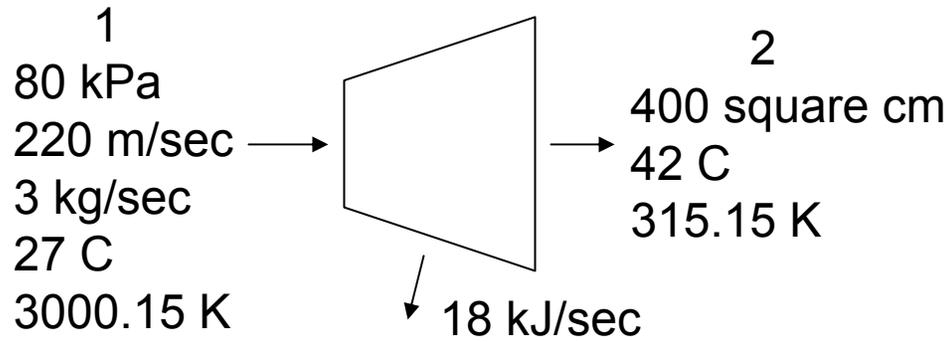


Problem Set 1 (PS1) Due Tuesday January 24

1. Air at 80 kPa, 27 C and 220 m/sec enters a diffuser at a rate of 3 kg/sec and leaves at 42 C. The exit area of the diffuser is 400 square cm. The air loses heat at a rate of 18 kJ/sec. Determine the exit velocity and the exit air pressure
2. In an insulated heat exchanger air is heated at atmospheric pressure from 25 C to 100 C and 2 kg/sec of water is cooled at atmospheric pressure from 150 C to 100 C. What is the entropy change for the air stream, the water stream and for the overall heat transfer process?
3. 3 lbs of air is expanded in an adiabatic isentropic process from 500 R and a gage pressure 200 psi to a gage pressure of 100 psi. Atmospheric pressure is 14.6 psi. What are the temperature, pressure, specific density and total volume at the end of the process ?

PS1.1



steady flow energy equation

$$m \left(c_p T_1 + \frac{u_1^2}{2 \times 1000} \right) + Q = m \left(c_p T_2 + \frac{u_2^2}{2 \times 1000} \right)$$

$$3 \text{ kg} \left(1.005 \times 300.15 \frac{220^2}{2000} \right) + 18 \text{ kJ/sec} = 3 \text{ kg} \left(1.005 \times 315.15 \frac{u_2^2}{2000} \right)$$

$$941 + 18 = 950.18 + .0015 u_2^2$$

$$u_2 = 76.7 \text{ m/sec}$$

continuity equation

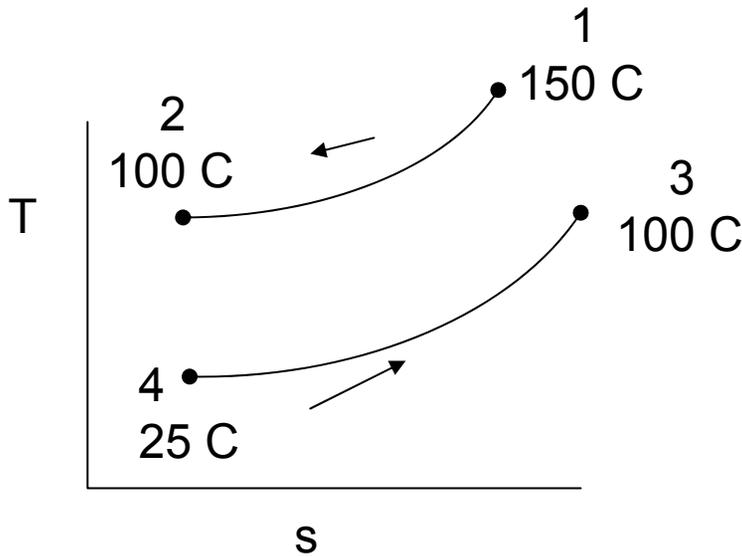
$$m = \rho_2 A_2 u_2$$

ideal gas law

$$m = \frac{p_2}{R T_2} A_2 u_2$$

$$p_2 = \frac{m R T_2}{A_2 u_2} = \frac{3 \times 8.324 \text{ kPa/kgmole K} \times 315.5 \text{ K}}{28.97 \times .04 \text{ m}^2 \times 76.68 \text{ m/sec}} = 88.46 \text{ kPa}$$

PS1.2



energy balance

$$Q_{\text{water}} = Q_{\text{air}}$$

$$m_{\text{water}} \times c_p \times (T_1 - T_2) = m_{\text{air}} \times c_p \times (T_1 - T_2)$$

$$m_{\text{air}} = \frac{2 \text{ kg} \times 4.18 \text{ kJ/kg K} \times (150 - 100)}{1.005 \text{ kJ/kg K} \times (100 - 25)}$$

$$m_{\text{air}} = 5.5 \text{ kg/sec}$$

entropy change

$$\Delta S_{\text{air}} = m_{\text{air}} c_p \ln\left(\frac{T_2}{T_1}\right) - R \ln\left(\frac{p_2}{p_1}\right)$$

$$\Delta S_{\text{air}} = 5.5 \times 1.005 \times \ln\left(\frac{373.15}{298.15}\right) = +1.228 \text{ kJ/K}$$

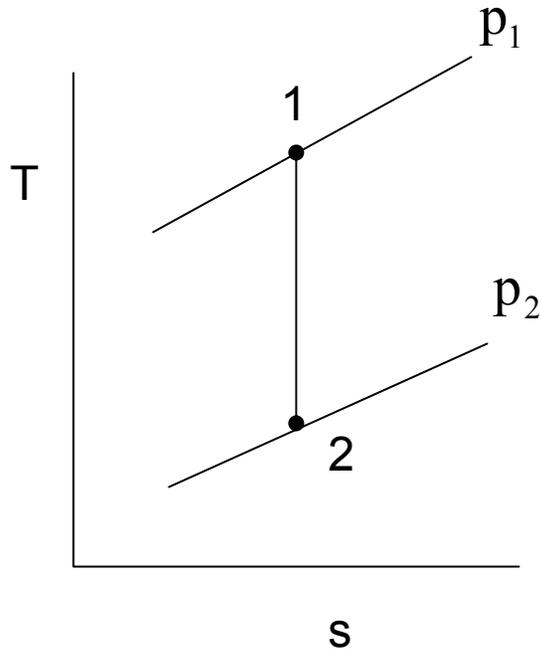
$$\Delta S_{\text{water}} = 2 \times 4.18 \times \ln\left(\frac{373.15}{423.15}\right) = -1.0512 \text{ kJ/K}$$

$$\Delta S_{\text{process}} = \Delta S_{\text{air}} + \Delta S_{\text{water}} = +1.228 - 1.0512$$

$$\Delta S_{\text{process}} = +.177 \text{ kJ/K}$$

$$\Delta S_{\text{isolated system}} \geq 0$$

PS 1.3



ideal gas law

$$\rho_1 = \frac{p}{RT} = \frac{214.6 \text{ lbf/in}^2 \times 144 \text{ in}^2/\text{ft}^2}{1545.15 \text{ ft lbf/lbmole} \times 500 \text{ R}} \times 28.97$$

$$\rho_1 = 1.158 \text{ lbm/ft}^3$$

isentropic process

$$\frac{T_2}{T_1} = \left(\frac{p_2}{p_1} \right)^{\frac{\gamma-1}{\gamma}} = \left(\frac{114.6}{214.6} \right)^{.2857} = .8359$$

$$T_2 = 500 \times .8359 = 417.96 \text{ R}$$

$$\frac{\rho_2}{\rho_1} = \left(\frac{p_2}{p_1} \right)^{\frac{1}{\gamma}} = \left(\frac{114.6}{214.6} \right)^{.7143} = .6388$$

$$\rho_2 = .6388 \times 1.158 = .7398 \text{ lbm/ft}^3$$

$$V_2 = \frac{m}{\rho} = \frac{3 \text{ lbm}}{.7398} = 4.055 \text{ ft}^3$$