

Second-Order Systems

The denominator of a second-order transfer function with $\zeta < 1$ can be expressed as form 4, Table 4.3. If $\zeta \geq 1$, the denominator can be written as the product of two first-order factors like form 3. A special case of a second-order system that does not fit into the previous cases occurs when $\zeta = \infty$. The form is

$$T(s) = \frac{K}{s(\tau s + 1)} \quad (4.5-37)$$

An example is a mass with a damper but no spring ($k = 0$). The three building blocks are K , s , and $\tau s + 1$. Because it is in the denominator, the s term shifts the composite m curve upward for $\omega < 1$ and shifts it down for $\omega > 1$. The composite m curve follows that of the s term until $\omega \approx 1/\tau$, when the $(\tau s + 1)$ term begins to have an effect. For $\omega \gg 1/\tau$, the composite slope is -40 db/decade. The s term contributes a constant -90° to the ϕ curve. The result is to shift the first-order lag curve (Figure 4.11b) down by 90° . The results are shown in Figure 4.18.

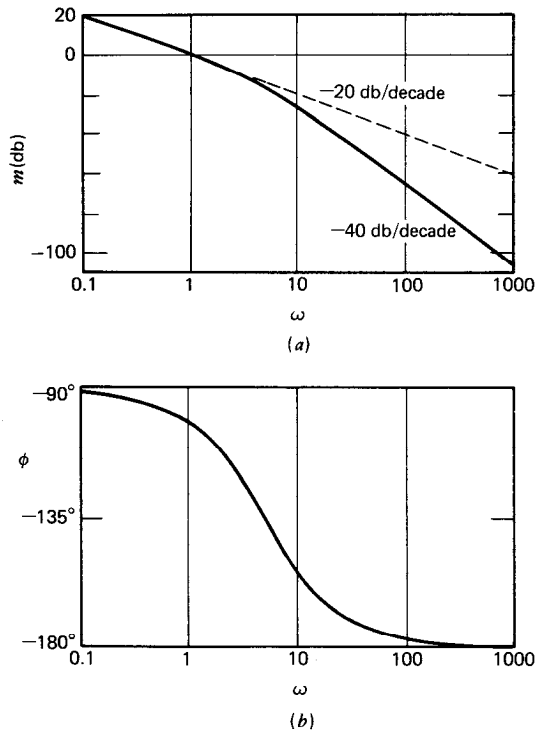


FIGURE 4.18 Frequency response plots for

$$T(s) = \frac{K}{s(\tau s + 1)}$$

with $K = 1$ and $\tau = 0.2$.

TABLE 4.3 Factors Commonly Found in Transfer Functions of the Form:

$$T(s) = K \frac{N_1(s)N_2(s) \dots}{D_1(s)D_2(s) \dots}$$

Factor $N_j(s)$ or $D_j(s)$

-
1. Constant, K
 2. s^n
 3. $\tau s + 1$
 4. $s^2 + 2\zeta\omega_n s + \omega_n^2 = \left[\left(\frac{s}{\omega_n} \right)^2 + \frac{2\zeta}{\omega_n} s + 1 \right] \omega_n^2, \quad \zeta < 1$
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An Overdamped System

Consider the second-order model

$$m\ddot{x} + c\dot{x} + kx = du(t)$$

Its transfer function is

$$T(s) = \frac{d}{ms^2 + cs + k} \quad (4.5-38)$$

If the system is overdamped, both roots are real and distinct, and we can write $T(s)$ as

$$T(s) = \frac{d/k}{\frac{m}{k}s^2 + \frac{c}{k}s + 1} = \frac{d/k}{(\tau_1 s + 1)(\tau_2 s + 1)} \quad (4.5-39)$$

where τ_1 and τ_2 are the time constants of the roots.

Returning to (4.5-22) and (4.5-23), we see that

$$m(\omega) = 20 \log |T(i\omega)| = 20 \log \left| \frac{d}{k} \right| - 20 \log |\tau_1 \omega i + 1| - 20 \log |\tau_2 \omega i + 1| \quad (4.5-40)$$

$$\phi(\omega) = \angle \frac{d}{k} - \angle (\tau_1 \omega i + 1) - \angle (\tau_2 \omega i + 1) \quad (4.5-41)$$

where $K = d/k$, $D_1(i\omega) = \tau_1 \omega i + 1$ and $D_2(i\omega) = \tau_2 \omega i + 1$. Thus, the magnitude ratio plot in db consists of a constant term, $20 \log |d/k|$, minus the sum of the plots for two first-order lead terms. Assume that $\tau_1 > \tau_2$. Then for $1/\tau_1 < \omega < 1/\tau_2$, the slope is approximately -20 db/decade. For $\omega > 1/\tau_2$, the contribution of the term $(\tau_2 \omega i + 1)$ is significant. This causes the slope to decrease by an additional 20 db/decade, to produce a net slope of -40 db/decade for $\omega > 1/\tau_2$. The rest of the plot can be sketched as before. The result is shown in Figure 4.19a for $d > k$. The phase angle plot shown in Figure 4.19b is produced in a similar manner by using (4.5-41). Note that if $d/k > 0$, $\angle(d/k) = 0^\circ$.

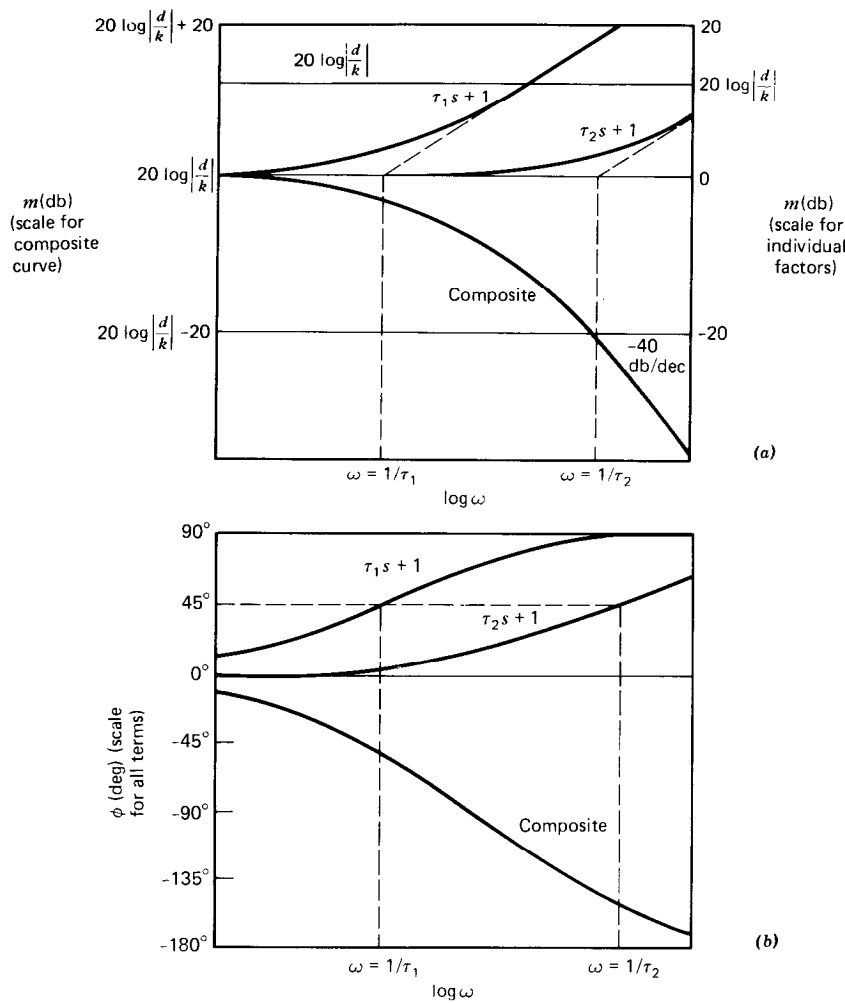


FIGURE 4.19 Frequency response plots for the overdamped system

$$T(s) = \frac{d/k}{(\tau_1 s + 1)(\tau_2 s + 1)}$$

An Underdamped System

If the transfer function given by (4.5-38) has complex conjugate roots, it can be expressed as form 4 in Table 4.3.

$$T(s) = \frac{d/k}{\frac{m}{k}s^2 + \frac{c}{k}s + 1} = \frac{d/k}{\left(\frac{s}{\omega_n}\right)^2 + 2\zeta\frac{s}{\omega_n} + 1} \quad (4.5-42)$$

We have seen that the constant term d/k merely shifts the magnitude ratio plot up or down by a fixed amount and adds either 0° or -180° to the phase angle plot. Therefore, for now, let us take $d/k = 1$ and consider the following quadratic factor, obtained from (4.5-42) by replacing s with $i\omega$.

$$T(i\omega) = \frac{1}{\left(\frac{i\omega}{\omega_n}\right)^2 + \frac{2\zeta}{\omega_n}i + 1} = \frac{1}{1 - \left(\frac{\omega}{\omega_n}\right)^2 + \frac{2\zeta\omega}{\omega_n}i} \quad (4.5-43)$$

The magnitude ratio is

$$\begin{aligned} m(\omega) &= 20 \log \left| \frac{1}{1 - \left(\frac{\omega}{\omega_n}\right)^2 + \frac{2\zeta\omega}{\omega_n}i} \right| \\ &= -20 \log \sqrt{\left(1 - \frac{\omega^2}{\omega_n^2}\right)^2 + \left(\frac{2\zeta\omega}{\omega_n}\right)^2} \\ &= -10 \log \left[\left(1 - \frac{\omega^2}{\omega_n^2}\right)^2 + \left(\frac{2\zeta\omega}{\omega_n}\right)^2 \right] \end{aligned} \quad (4.5-44)$$

The asymptotic approximations are as follows. For $\omega \ll \omega_n$,

$$m(\omega) \cong -20 \log 1 = 0$$

For $\omega \gg \omega_n$,

$$\begin{aligned} m(\omega) &\cong -20 \log \sqrt{\frac{\omega^4}{\omega_n^4} + 4\zeta^2 \frac{\omega^2}{\omega_n^2}} \\ &\cong -20 \log \sqrt{\frac{\omega^4}{\omega_n^4}} \\ &= -40 \log \frac{\omega}{\omega_n} \end{aligned}$$

Thus, for low frequencies, the curve is horizontal at $m = 0$, while for high frequencies, it has a slope of -40 db/decade, just as in the overdamped case. The high-frequency and low-frequency asymptotes intersect at the corner frequency $\omega = \omega_n$.

The underdamped case differs from the overdamped case in the vicinity of the corner frequency. To see this, examine $M(\omega)$.

$$M(\omega) = \frac{1}{\sqrt{\left(1 - \frac{\omega^2}{\omega_n^2}\right)^2 + \left(\frac{2\zeta\omega}{\omega_n}\right)^2}} \quad (4.5-45)$$

This has a maximum value when the denominator has a minimum. Setting the derivative of the denominator with respect to ω equal to zero shows that the maximum $M(\omega)$ occurs at $\omega = \omega_n \sqrt{1 - 2\zeta^2}$. This frequency is the *resonant frequency* ω_r . The peak of $M(\omega)$ exists only when the term under the radical is positive; that is, when $\zeta \leq 0.707$. Thus,

$$\omega_r = \omega_n \sqrt{1 - 2\zeta^2} \quad 0 \leq \zeta \leq 0.707 \quad (4.5-46)$$

The value of the peak M_p is found by substituting ω_r into $M(\omega)$. This gives

$$M_p = M(\omega_r) = \frac{1}{2\zeta\sqrt{1-\zeta^2}} \quad 0 \leq \zeta \leq 0.707 \quad (4.5-47)$$

If $\zeta > 0.707$, no peak exists, and the maximum value of M occurs at $\omega = 0$ where $M = 1$. Note that as $\zeta \rightarrow 0$, $\omega_r \rightarrow \omega_n$, and $M_p \rightarrow \infty$. For an undamped system, the resonant frequency is the natural frequency ω_n .

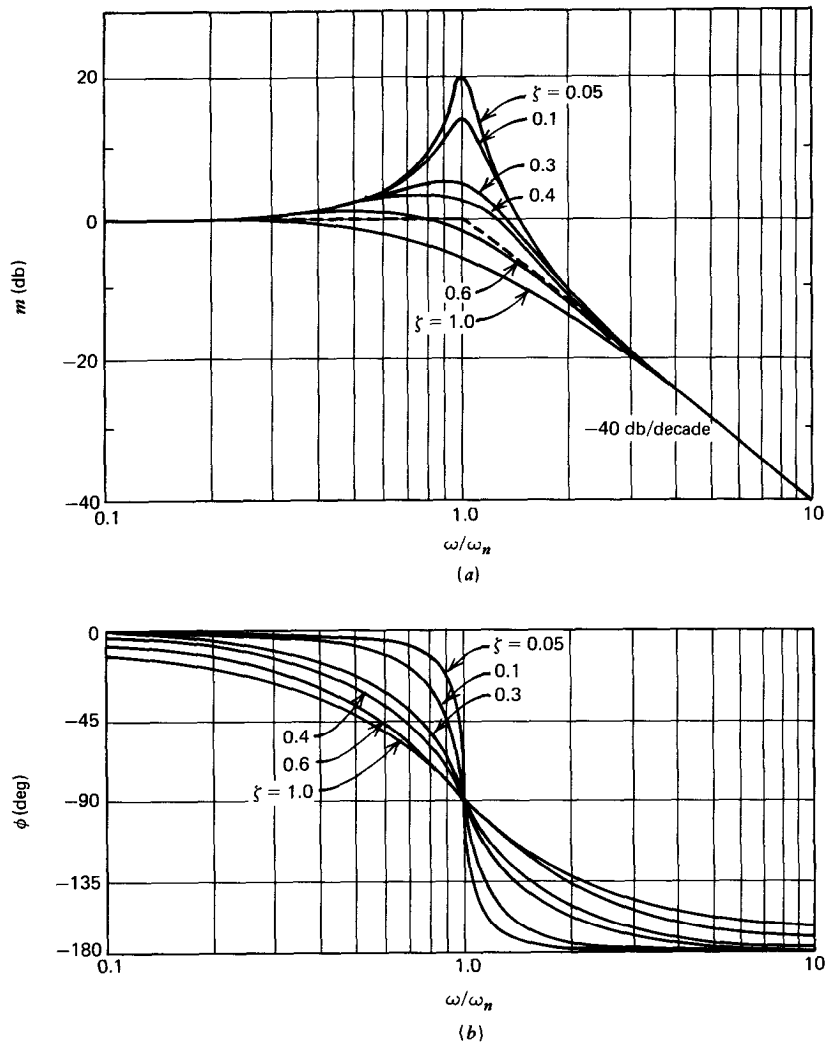


FIGURE 4.20 Frequency response plots for the underdamped system

$$T(s) = \frac{1}{\left(\frac{s}{\omega_n}\right)^2 + \frac{2\zeta s}{\omega_n} + 1}$$

A plot of $m(\omega)$ versus $\log \omega$ is shown in Figure 4.20a for several values of ζ . Note that the correction to the asymptotic approximations in the vicinity of the corner frequency depends on the value of ζ . The peak value in decibels is

$$m_p = m(\omega_r) = -20 \log (2\zeta \sqrt{1 - \zeta^2}) \quad (4.5-48)$$

At $\omega = \omega_n$,

$$m(\omega_n) = -20 \log 2\zeta \quad (4.5-49)$$

The curve can be sketched more accurately by repeated evaluation of (4.5-44) for values of ω near ω_n .

The phase angle plot is obtained in a similar manner. From the additive property for angles (4.5-23), we see that for (4.5-43),

$$\phi(\omega) = -\angle \left[1 - \left(\frac{\omega}{\omega_n} \right)^2 + \frac{2\zeta\omega}{\omega_n} i \right]$$

Thus

$$\tan \phi(\omega) = - \left[\frac{\frac{2\zeta\omega}{\omega_n}}{1 - \left(\frac{\omega}{\omega_n} \right)^2} \right] \quad (4.5-50)$$

where $\phi(\omega)$ is in the 3rd or 4th quadrant. For $\omega \ll \omega_n$,

$$\phi(\omega) \cong -\tan^{-1} 0 = 0^\circ$$

For $\omega \gg \omega_n$,

$$\phi(\omega) \cong -180^\circ$$

At the corner frequency,

$$\phi(\omega_n) = -\tan^{-1} \infty = -90^\circ$$

This result is independent of ζ . The curve is skew-symmetric about the inflection point at $\phi = -90^\circ$ for all values of ζ . The rest of the plot can be sketched by evaluating (4.5-50) at various values of ω . The plot is shown for several values of ζ in Figure 4.20b. At the resonant frequency,

$$\phi(\omega_r) = -\tan^{-1} \frac{\sqrt{1 - 2\zeta^2}}{\zeta} \quad (4.5-51)$$

For our applications, the quadratic factor given by form 4 in Table 4.3 almost always occurs in the denominator; therefore, we have developed the results assuming this will be the case. If a quadratic factor is found in the numerator, its values of $m(\omega)$ and $\phi(\omega)$ are the negative of those given by (4.5-44) and (4.5-50).